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ROTOR BLADE FOR A WIND-DRIVEN POWER-PLANT

BACKGROUND OF THE INVENTION

FIELD OF THE INVENTION

[0001] The present invention relates to a rotor blade for a power-plant fitted with a

device optimizing aerodynamics.

DESCRIPTION OF RELATED ART

[0002] Wind-driven power-plants comprise a rotor fitted with one or more rotor

blades. Each rotor blade in turn comprises a sectional blade contour, which is per-

pendicular to the blade's direction from blade root to blade tip, and of which the

thickness decreases toward the outside, from the blade root toward the blade tip.

The blade contour entails a suction side and a pressure side, so that when air is

moving around the blade, the suction side is at a lower pressure than the pressure

side. The pressure differential across the pressure and suction sides generates lift

causing the rotor rotation, which in turn is used to drive an electric power generator.

[00031 High rotor efficiency and hence high wind-driven power-plant output as-

sumes, as smooth as possible, an airflow around the blade contour perpendicularly

to the axis of the rotor blade and over the entire range of the rotor blades.

[0004] However, it is noted with respect to known rotor blades that the airflow

moving around the blade contour will detach at the suction side and that a wake

zone is generated which, by increasing drag, reduces the rotor blade lift and decel-

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erates the rotor blade. Typically, the term "wake zone" denotes the region of the de-

tached airflow. Both factors lower the wind-driven power-plant productivity.

[0005] Seen in the direction of flow, airflow detachment as a rule takes place be-

yond the highest relative depth of the blade contour. In general, at least the zones

near the blade root will be affected.

[0006] Vortex generators are a known means to reduce airflow detachment and

thus to optimize aerodynamic airflow around the blade contour. Such generators, as

a rule, are in the form of sheetmetal, bars or cross-sectionally shaped structures and

the like configured at the suction side of the rotor blade and generating local turbu-

lences reducing large-area detachment of the airflow around the blade contour. Il-

lustratively, such vortex generators are known form the patent document WO

0015961. These known vortex generators incur the drawback that they only slightly

improve wind-driven power-plant productivity because they themselves generate

drag and furthermore are very noisy.

BRIEF SUMMARY OF THE INVENTION

[0007] The objective of the present invention is to create a wind-driven power-

plant rotor blade that offers substantially improved aerodynamic airflow around the

blade.

[0008] The solution to this problem is comprehensively discussed below and is

based on the insight that interfering flows are generated, in particular, in the zone of

the rotor root at the suction side of the blade contour and run transversely to the ro-

tor blade toward the blade tip. These cross-flows arise substantially in the region of

the detached airflow and are assumed due to the pressure differentials caused by

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different incident airflow speeds at different blade radii; they are induced in an espe-

cially marked manner in the blade root zone. In addition to these factors, the cen-

trifugal forces acting on the rotor blade also contribute to generating said cross-

flows.

[0009] The detachment in the rotor-blade root zone is moved by the cross-flow

out of the airflow around the aerodynamically disadvantageous blade root zone to-

ward the blade tip, that is into the zone of the aerodynamically more advantageous

blade contours. Moreover the cross-flow also interferes with the effective airflow

around the rotor blade because of the generation of turbulences that entail prema-

ture airflow detachment.

[0010] Accordingly, the present invention provides a wind-driven power-plant ro-

tor blade fitted with a device optimizing the airflow around the blade contour, the de-

vice comprising at least one planar element mounted by one of its thin sides on the

suction side and substantially pointing in the direction of airflow. The element is con-

figured in the zone, the above cited cross-flow running outward on the suction side of

the contour from the blade root, and the height and length of the device being se-

lected so that it shall substantially reduce the cross-flow.

[0011] The reduction of the cross-flow by the planar element prevents premature

airflow detachment from the rotor blade suction side. Such improved blade-

enveloping airflow achieves a considerable increase in the output of a correspond-

ingly equipped wind-driven power-plant without entailing an increase in operational

noise.

[0012] The required height and length of the particular planar element and the op-

timal position of this element on the suction side of the rotor blade inherently varies

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with the distance from the rotor axis of rotation, the blade contour depth, the rotor

width, the most likely speed of the incident airflow, etc.

[0013] In the simplest way, the best configuration is ascertained empirically, for

instance by affixing rows of wool threads at one end to the rotor blade and by field

tests visually determining the prevailing airflow conditions by means of the said

threads' free ends. In this manner the effect of planar elements of the invention on

the airflow conditions may be ascertained relatively easily in various radius positions

and consequently, also, the optimal number and positions, and if called for, also the

sizes of the planar elements of the invention.

[0014] The wool threads may be configured, if additionally called for, on differ-

ently sized spacers, in the form of bars for instance, to determine the wake zone

depth caused by the cross-flow and hence the height of the cross-flow to be stopped.

[0015] This procedure allows for empirically ascertaining the optimal height and

length of the planar elements of the invention and/or their optimal positions on a par-

ticular rotor blade. Running corresponding series of tests, the optimal dimensions

and positions of the planar elements of the invention may be determined for arbitrary

types of rotor blades.

[0016] Planar elements preventing cross-flows mounted on the suction side of an

airfoil have long been known in aeronautical engineering. These elements are espe-

cially widely used in swept-back wing aircraft. In this design, the problem arises that

on account of the obliqueness of the leading airfoil edge, a pressure gradient is gen-

erated which deflects the air flowing around the airfoil toward the airfoil tip. This un-

detached cross-flow in turn interferes with the airflow around the airfoil and hence

reduces the lift because the airflow moves along the wing, but no longer on it. To

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reduce the cross-flow, therefore, perpendicular barriers are affixed to such airfoils

and are denoted as boundary layer fences.

[0017] These boundary layer fences differ as regards their essential features from

the above-cited planar elements of the invention for wind-driven power-plant rotor

blades. The undetached cross-flows at sweptback airfoils being induced foremost in

the region of the airfoil leading edge, this is precisely the area where the boundary

layer fences are erected. Frequently, the fences even run around the airfoil leading

edge as far as to the airfoil pressure side.

[0018] The wind-driven power-plant's rotor blade planar elements of the inven-

tion, on the other hand, reduce a cross-flow which was caused by other phenomena

and which already has detached and which arises predominantly in the region of

maximum depth of rotor blade contour and induces airflow detachment in the region

of the maximum depth. Configuring such elements merely in the region of the rotor

blade leading edge would be inappropriate.

[0019] In one preferred embodiment of the present invention, the planar element

is configured at least in the zone of the cross-flow running on the blade contour suc-

tion side between a zone of maximum relative blade depth and blade trailing edge.

This cross-flow is the above described flow that was generated by the speed differ-

ential of the incident airflow between the zones near the rotor blade root and the

zones near the blade tips and by the resulting pressure gradients at the rotor blade

suction side and also the centrifugal forces at the rotor blade.

[0020] In an especially preferred embodiment mode of the present invention, the

planar elements extend over the full width of the rotor blade suction side. In this

manner, encroaching by the cross-flow into zones of proper airflows is precluded

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even without knowledge of the accurate path of the cross-flow on the rotor blade

suction side.

[0021] In a further preferred embodiment of the invention, the planar element is

designed so that its length runs straight. In this manner the planar element's drag is

kept small and noise is minimized. In an especially preferred embodiment mode, the

planar element runs no more than 10° away from the tangent to that circle formed by

the rotor blade radius which is subtended by the planar element position.

[0022] In a further preferred embodiment of the invention, the planar element is

designed so that its longitudinal direction follows the radius path corresponding to

the distance between the planar element's front end and the rotor's axis of rotation.

In turn, this design makes it possible to minimize the element's drag and its noise.

[0023] In a further preferred embodiment of the invention, the rotor blade is fitted

at the suction side of its contour with several planar elements. This design is appro-

priate when, behind the first element, a new cross-flow as discussed herein shall be

produced. Optimal positioning and sizing of these several planar elements on the

rotor blade can be implemented as described above.

[0024] In a further preferred embodiment of the invention, planar elements are

mounted on the rotor blade suction side in a zone extending from the blade root to

half the rotor blade's length. In especially preferred manner, the zone shall extend

from the blade root to one third the rotor blade's length.

[0025] In an especially preferred embodiment of the invention, at least one planar

element is mounted in a zone which, as seen from the blade root, is situated beyond

a transition range wherein the sectional contour of the blade root merges into a lift

generating contour. An element mounted in such a zone is appropriate, for instance,

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to interrupt an already extant cross-flow coming from the blade root zone and in this

manner to eliminate interferences from laminar airflows in this zone.

[0026] In another preferred embodiment mode, at least one planar element is

mounted in a zone extending from the blade root to the near side of a transition

range where the blade root contour merges into a lift-generating contour. Because

of the special conditions relating to rotor blades, the substantial portion of the inter-

fering cross-flow arises in this zone near the blade root because this rotor blade root,

on account of its contour depth, as a rule is not an aerodynamically advantageous

contour. Accordingly, the configuration of an element of the invention in this zone

especially effectively suppresses the formation of cross-flows -- contrary to the case

of the above described elements which are configured in a manner that they prevent

propagation of an already extant cross-flow into the power-delivering rotor blade

zone.

[0027] In a further advantageous embodiment of the invention, the planar element

is air-permeable at least segment-wise, for instance being a mesh or perforated. A

planar element of this design, when appropriately dimensioned, may better reduce

cross-flows than a continuous planar element. Also, such a design reduces the pla-

nar element weight.

[0028] In a further embodiment mode of the invention, the planar element is made

of a metal, for instance high-grade steel or aluminum, of plastic, of compound mate-

rials such as GRP (glass-fiber reinforced plastic) or CFP (carbon-fiber reinforced

plastic), or of a combination of such materials. Such a design assures that the ele-

ment shall both be weather-proof and withstand the operational mechanical loads. It

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is understood that other materials may be employed in an equivalent manner pro-

vided they meet the conditions of weathering and mechanical strength.

[0029] The invention applies not only to rotor blades but also to planar elements

that may be aligned in a sectionally contoured wind-driven power-plant rotor blade

substantially in the direction of the airflow and at a spacing from the suction side,

their height and length being selected in a way that they shall implement an effective

reduction of a cross-flow running outward from the blade root. These elements may

also be used to retrofit already erected wind-driven power-plants.

[0030] A preferred embodiment provides that the planar element be configured to

be tightly adjoined to the contour of the rotor section. However, the element also

may be deformed in an elastic or plastic manner to allow it to be matched to the con-

tour of the rotor section at the first instance it is assembled to it.

[0031] The present invention is shown in illustrative and schematic manner in the

drawings of a preferred embodiment.

BRIEF DESCRIPTION OF THE DRAWINGS

[0032] Fig. 1 is the top view of the suction side of a wind-driven power-plant rotor

blade,

[0033] Fig. 2a is a section along line A-A of Fig. 1, and

[0034] Fig. 2b is a further section along line A-A of Fig. 1 in another embodiment

mode.

DETAILED DESCRIPTION OF THE INVENTION

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[0035] Fig. 1 shows a rotor blade 10 having a leading edge 11, a trailing edge 12,

a blade root 13, a blade tip 14, a suction side 15 and a pressure side 16. The rela-

tive thickness of the rotor blade decreases towards the outside from the blade root

13 to the blade tip 14. The leading edge 11 points in the direction of rotation of the

rotor blade. Planar elements 17 and 18 are mounted in the direction of airflow at the

suction side 15 and suppress cross-flows on the suction side 15 and preclude pre-

mature airflow detachment. A transition range 19 is characterized in that the contour

of the blade root 13, which in this instance is cylindrical, merges into a pear-shaped,

lift-generating contour. An arrow denotes the cross-flow.

[0036] Fig. 2a shows a section of the rotor blade along line A-A of Fig. 1, the

blade having a leading edge 21, a trailing edge 22, a suction side 25 and a pressure

side 26. The planar element 27 is mounted on the suction side 25 and runs from the

leading edge 21 to the trailing edge 22 and suppresses cross-flows on the suction

side 25.

Г00371 Fig. 2b shows a further section of the rotor blade along line A-A in Fig. 1

for another embodiment mode, the blade having a leading edge 21, a trailing edge

22, a suction side 25 and a pressure side 26. The planar element 28 is mounted on

the suction side 25 and runs from the leading edge 21 to the trailing edge 22, and is

fitted with rounded edges. The planar element 28 is fitted with perforations 29 which,

when appropriately sized, contribute to effectively suppress cross-flows on the suc-

tion side 25.

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